

# **HEAT PUMPS**

**NET4ENERGY** 



# **NET4ENERGY**

## **Partners**







## History of heat pumps

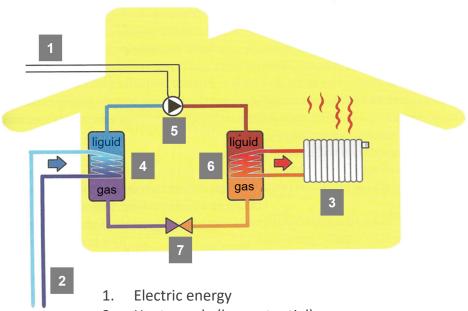


prof. Ing. Aurel Stodola, Dr. h. c.

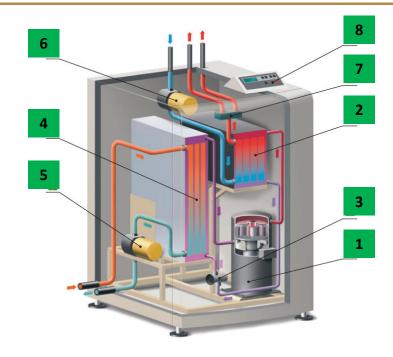
(\* <u>11th May 1859</u>, <u>Liptovský Mikuláš</u> – SLOVAKIA † <u>25th December 1942</u>, <u>Zürich</u>, <u>Switzerland</u>)

He was the designer of the first heat pump. His heat pump from 1928 still works in Switzerland and heats the town hall in Geneva with heat removal from the lake water (it is a closed circuit).

## Principle of heat pump operation

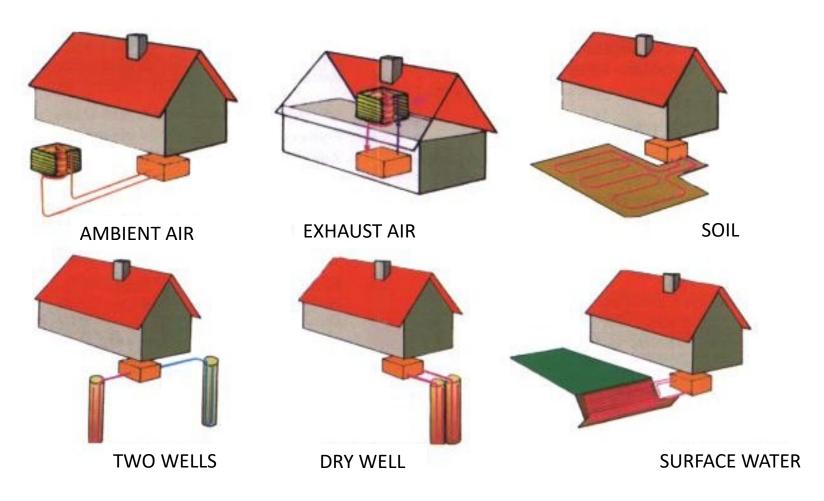


- 2. Heat supply (low potential)
- 3. Heat transfer (high potential)
- 4. Evaporator (source circuit plate heat exchanger)
- 5. Compressor
- 6. Capacitor (plate heat exchanger heating circuit)
- 7. Expansion valve

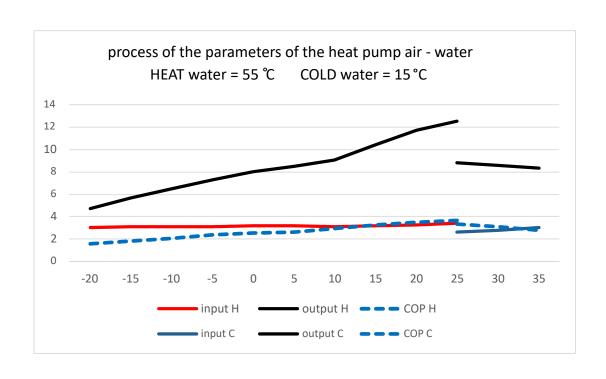


- 1. Compressor
- 2. Capacitor (plate heat exchanger heating circuit)
- 3. Expansion valve
- 4. Evaporator (source circuit plate heat exchanger)
- 5. Circulation pump (source circuit)
- 6. Heating circuit pump
- 7. Switching valve (heating/domestic hot water)
- 8. Control panel

# Possibilities of obtaining low-potential heat

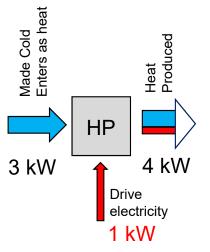


## Energy efficiency of a heat pump



## COP = coefficient of performance

In heat transformation, energy efficiency is characterized by coefficient of performance.

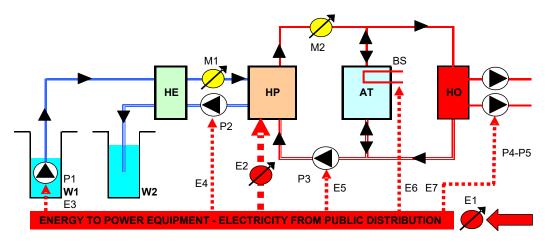


COP heating = 
$$\frac{4}{1}$$
 = 4

COP cooling = 
$$\frac{3}{1}$$
 = 3

COP h+c = 
$$\frac{4+3}{1}$$
 = 7

## Energy efficiency of a heat pump



#### LEGEND:

HEAT EXCHANGER

HEAT PUMP WATER / WATER

M1-2 THERMOMETER

**E1-2** ELECTRICITY METER

HEAT ACCUMULATION TANK

HEATED OBJECT

BIVALENT / BACKUP HEAT SOURCE

W1-2 SOURCE SOURCE / INTAKE WELL

HEAT EXCHANGER

SUBMERSIBLE PUMP

P2-3 CIRCULATION MACHINE PUMPS

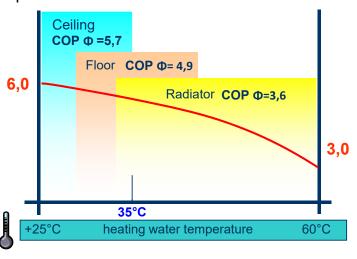
P4-5 CIRCULATING OBJECT PUMPS

$$COP = \frac{M2}{E2} = \frac{M1+E2}{E2}$$
  $SPF = \frac{M2}{E1} = \frac{M2}{E2+E3+E4+E5+E6+E7}$ 

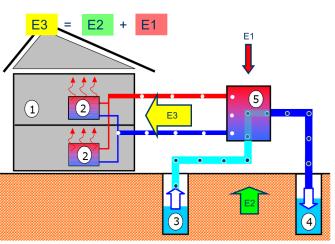
### SPF = seasonal performance faktor

For the year-round efficiency of the heat pump heat transformation system, energy efficiency is characterized by a seasonal performance factor

Influence of the heating system on the average COP of the heat pump







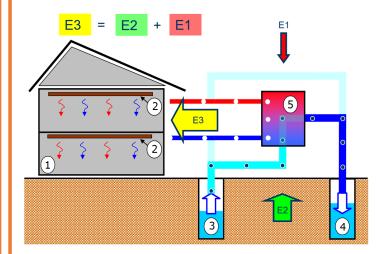
$$SPF = \frac{E3}{E1} = cca 3,1$$

#### **LEGEND:**

- 1 BUILDING HEATING
- **2** RADIATORS
- **3** SUCTION WELL (source of heat and cold)
- **4** INFILTRATION WELL
- **5 HEAT PUMP** WATER-WATER heat source

## **HEATING**

ceiling radiant system



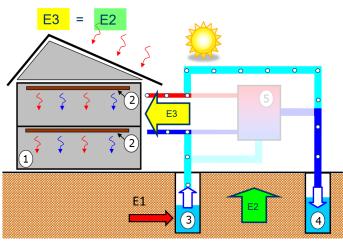
$$SPF = \frac{E3}{E1} = cca 4,5$$

#### **SPF** = seasonal performance faktor

- **E1** total supplied electricity for propulsion
- **E2** heat obtained from the well
- E3 heat produced delivered to the building

### **PASIVE COOLING**

ceiling radiant system



$$SPF = \frac{E3}{E1} = cca 7 - 10$$

#### **LEGEND:**

- 1 BUILDING HEATING / COOLING
- **2** CEILING RADIANT SYSTEM
- 3 SUCTION WELL (source of heat and cold)
- **4** INFILTRATION WELL
- **5 HEAT PUMP** WATER-WATER heat source

# Types of air heat pumps SPLIT

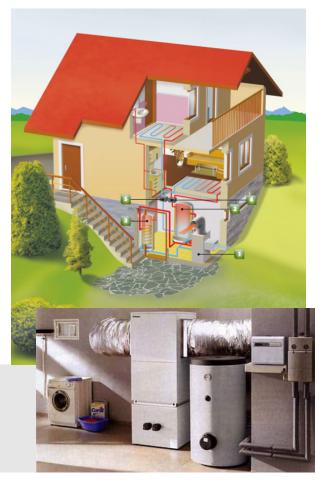


- 1. Heat pump
- 2. Separate air evaporator
- 3. Heat storage tank for heating
- 4. Hot water tank
- 5. Distributor of floor heating
- 6. Collector of underfloor heating

#### **SOURCE - AIR**

- Original heat pumps under -7°C did not heat current until -20°C
- After 3500 m³/h freezing occurs (cca 6 8 minutes the defrost heat is turned – condensate drain = 30 - 100 liters/day)
- Suitable for solution via accumulation tanks
- Sensitive to accumulation tank size (the size of the accumulation tank affects the switching of the heat pump) lifetime of the heat pump
  - 1. Compact heat pump air/water
  - 2. Heat storage tank for heating
  - 3. Hot water tank
  - 4. Distributor of floor heating
  - 5. Collector of underfloor heating

#### **COMPACT**



## Types of water heat pumps

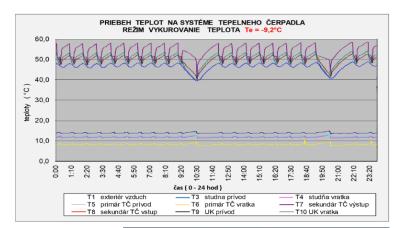
Water / Water from well

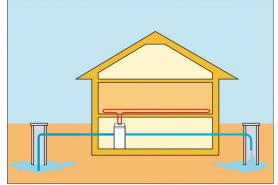


- 1. Heat pump
- 2. Source well with submersible pump
- 3. Suction well
- 4. Heat storage tank for heating
- 5. Hot water tank
- 6. Distributor of floor heating
- 7. Collector of underfloor heating

# SOURCE - WELL MOST BENEFICIAL OPERATION

- COVERAGE OF WATER 3-5 m<sup>3</sup>/hour for 10 15 kW (pumping test 1 month,...) we must return the water
- WATER TEMPERATURE cca 10°C usability up to +7°C and is blocked at +5°C
- WATER COMPOSITION (chemicals must not be flocculated in the water)





## Types of ground source heat pumps

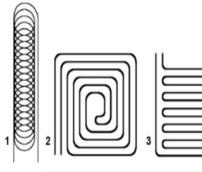
### Flat ground collector



- 1. Heat pump
- 2. Flat ground collector
- 3. Distributor and collector of ground collector
- 4. Heat storage tank for heating
- 5. Hot water tank
- 6. Distributor of floor heating
- 7. Collector of underfloor heating

#### **SOURCE - FLAT GROUND COLLECTOR**

- LAYING DEPTH 1,2 1,5 m
- WITHDRAWAL OF PIPES 0,5 1,0 m
- HIGH PUMPING WORK loop max. 150 m
- IMPROPER INSTALLATION REDUCES EFFICIENCY
- CONNECTION TO CIRCUITS inlet return = MORE CIRCUITS
- DEEP ROOT PLANTS DO NOT GROW trees
- THE VEGETATION PERIOD IS MOVED BY cca 2 months
- SPECIFIC SOIL POWER:
  - spacing 0.6 1.0 m depth 1.2 1.5 m lenght 100 m cca  $20 \text{ W/m}^2$
  - Dry sandy soil
  - Do not place under the object





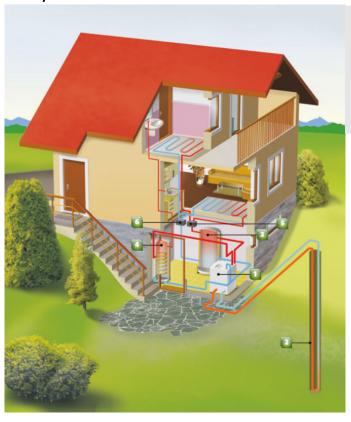
- 2. Špirálové uloženie
- 3. Meander





## Types of ground source heat pumps

Dry well



- 1. Heat pump
- 2. Dry well
- 3. Heat storage tank for heating
- 4. Hot water tank
- Distributor of floor heating
- 6. Collector of underfloor heating





#### **SOURCE - DRY WELL**

- DO NOT DRILL UNDER THE OBJECT distance of at least 2 m due to pressurized water
  - 15 m underground is no longer affected by surface temperature
- DRILL SPACING is about 10% of their depth (the problem of drilling vertically the possibility of crossing drills,...)
- Optimal drilling depth 100-120 m (deeper drilling has larger pumping works)
- The specific power of the drill is cca 55 W/m
- Possibility to use the dry well for cooling in summer

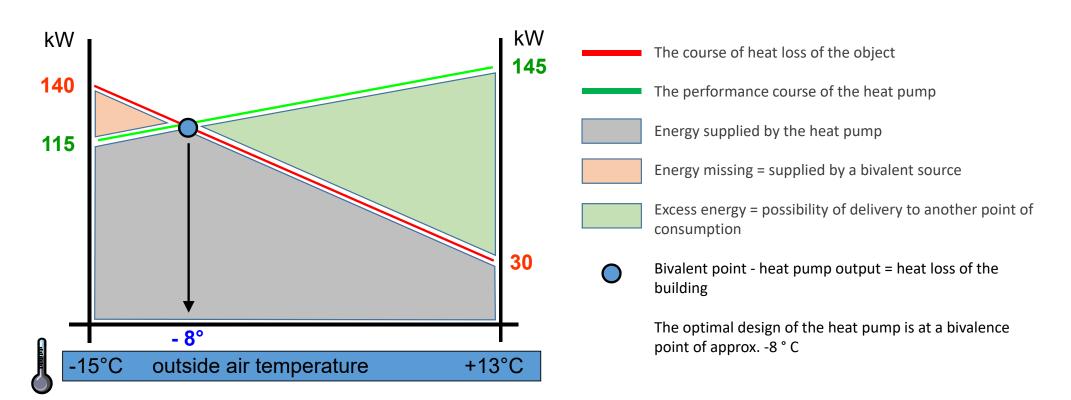
Groundwater dew point problem= 12°C (EXAMPLE room 25°C humidity 60% = dew point cca 15°C)

Difference max 6°C between air and ceiling temperature

- Heating DHW takes cca 1 hour when the HEATING or COOLING is stopped
- Cooling capacity is cca 1/3 from heat losses

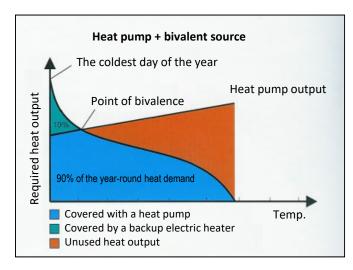
## Method of designing heat pumps

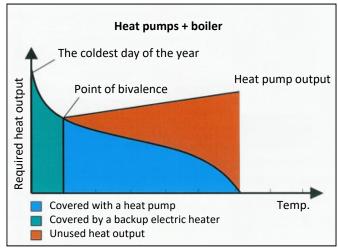
The course of heat losses of the building and the course of heat pump output

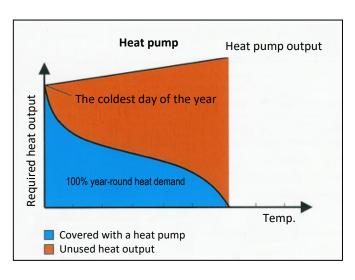


## Method of designing heat pumps – DEMONSTRATION OF SIMULATIONS effect on COP and SPF

Depending on how the heat pump and bivalent heat source are used







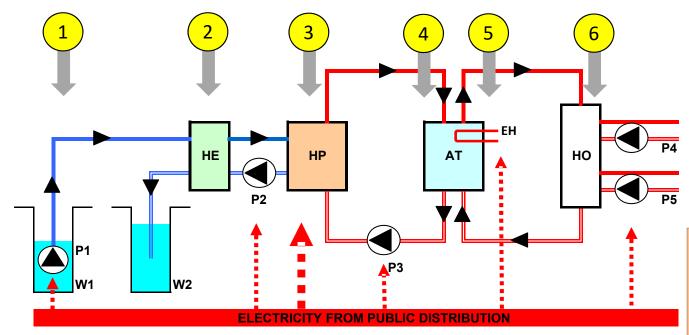
The entire temperature range is provided by the heat pump + the missing power is supplemented by a bivalent heat source

The heat pump provides performance only up to the bivalence point. Below the bivalence point temperature, the entire heating output is provided by the boiler

The heat pump provides the entire output without the need for a bivalent heat source

## Principles of design of heat pump system elements that affect SPF

### Water/Water from well



#### LEGEND:

HE	HEAT EXCHANGER
HP	HEAT PUMP WATER / WATER
ΑT	ACCUMULATION TANK

HO HEATED OBJECT

W1 SUCTION WELL (SOURCE)

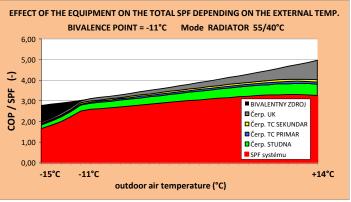
W2 IMPACT WELL

**P** PUMP

**EH** ELECTRIC HEATER

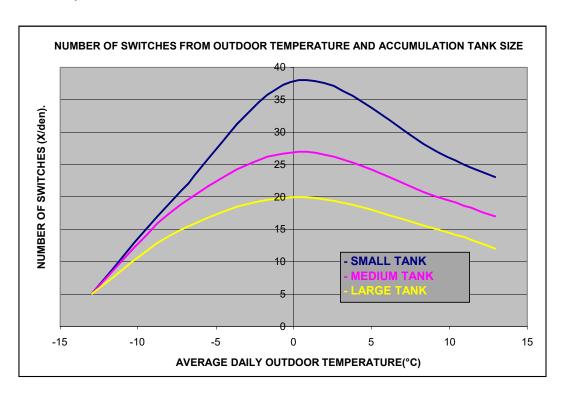
# HEAT PUMP SYSTEM ELEMENTS ASSESSED DEFINITION OF PRINCIPLES - EXAMPLE OF MEASUREMENTS

- 1. Source well with submersible pump
- 2. Heat exchanger
- 3. Heat pump
- 4. Heat storage tank
- 5. Bivalent heat source
- 6. Sampling point heating system



# Energy storage – dependence of the number of switching on the heat pump on the size of the storage tank

Water/Water from well



#### **DEMONSTRATION OF MEASUREMENTS AND SIMULATIONS**

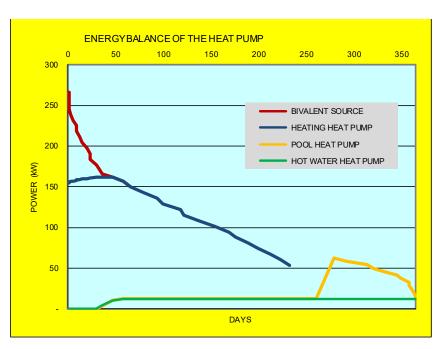
- 1. Heat pump
- 2. Source well with submersible pump
- 3. Suction well
- 4. Heat storage tank for heating
- 5. Hot water tank
- 6. Distributor of floor heating
- 7. Collector of underfloor heating

# Energy balance – DEMONSTRATION OF SIMULATIONS

Water/Water from well

#### **Family house**

- 1. Heating 10 kW
- 2. Domestic hot water 4 persons
- 3. Outdoor pool 25 m2 operation 90 days



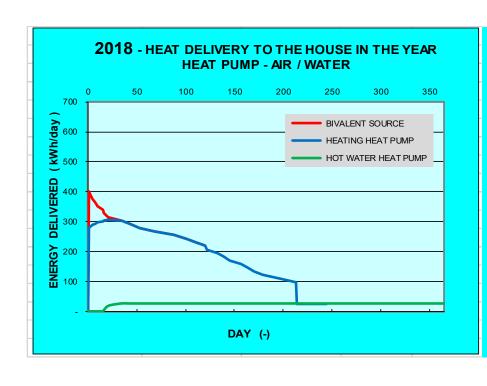
HEATING HEAT	25 605	kWh	77.3%					
HEAT WATER HEAT	4 457	kWh	13.4%					
MADE HEAT	33 138	kWh	100%					
HEAT PUMP HEAT	28 621	kWh	86.4%					
BIVALENT SOURCE HEAT	1 442	kWh	4.4%					
TOTAL HEAT	33 138	kWh	100%					
ELECTRICITY FOR HEAT P	5 147	kWh	77.9%					
FOR SUPPORTING EQU	897	kWh	13.6%					
FOR BIVALENT SOURC	1 442	kWh	21.8%					
TOTAL ELECTRICITY	6 610	kWh	100%					
	•							
HEATING	COP =	6.05	SPF =	4.16				
HOT WATER	COP =	3.72	SPF =	3.30				
SYSTEM	COP =	5.56	SPF =	4.02				

## Energy balance – DEMONSTRATION OF SIMULATIONS

Air/Water

#### **Apartment building**

- 1. Heating 18 kW
- 2. Domestic hot water 10 persons

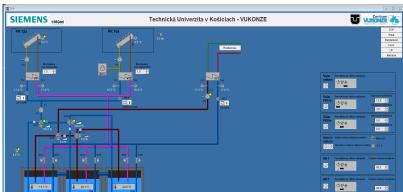


HEATING HEAT	40 593	kWh	80.5%	
HEAT WATER HEAT	9 833	kWh	19.5%	
MADE HEAT	50 426	kWh	100%	
HEAT PUMP HEAT	49 366	kWh	97.9%	
BIVALENT SOURCE HEAT	1 059	kWh	2.1%	
TOTAL HEAT	50 426	kWh	100%	
ELECTRICITY FOR HEAT PUMP	16 094	kWh	93.8%	
FOR SUPPORTING EQUIPM	0	kWh	0.0%	
FOR BIVALENT SOURCE	1 059	kWh	6.2%	
TOTAL ELECTRICITY	17 153	kWh	100%	
HEATING	COP =	3.04	SPF =	2.91
HOT WATER	COP =	3.18	SPF =	2.74
SYSTEM	COP =	0.00	SPF =	2.94

## **NESICA**

## Main activities of the project – Slovakia - excursions





**SOLAR** 









#### **VUKONZE**

- Hot water solar collectors
- Long-term accumulation of energy in water
- Photovoltaic panels
- Accumulation of electrical energy in batteries

#### **EcoPoint**

- Deep earth drilling
- Long-term accumulation of energy in the ground
- Heat Cold heat pump, accumulating concrete core (heating/cooling)

#### **TECHNIKOM**

- Wells – active and passive cooling, heat pumps, energy recovery by heat pumps

# Thank you for your attention